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Experiments with a One Parameter Computed Torque Position Control Method Using a PLC and a Simple Linear Actuator

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Linear positioning testbed

- Cart moved by a brushed DC motor along a linear rail
- Motor driver is connected to PLC analog output
- Can be converted to pendulum on a cart system
- As many robust industrial components, as possible
- Pos. Resolution: 0.13 mm

rmotor Limit switch

- Clogging estimated as ± 0.2 -0.3 mm
- Min. cycle time: 1 ms

Motor pulle



Closed-loop CTC (traditional method)

$$\begin{split} e(t) &= q^{N}(t) - q(t) & \dot{e}(t) = \dot{q}^{N}(t) - \dot{q}(t) \\ & \stackrel{\text{Derivative gain}}{\overset{\text{Derivative gain}}{\overset{\text{Proportional gain}}{\overset{\text{Proportional$$

 $\ddot{q}(t) = \ddot{q}^{Des}(t)$

Closed-loop CTC (traditional method, stability)

$$\ddot{q}(t) = \ddot{q}^{N}(t) + K_{D}\dot{e}(t) + K_{P}e(t)$$
$$\ddot{q}^{N}(t) - \ddot{q}(t) + K_{D}\dot{e}(t) + K_{P}e(t) = 0 \implies$$
$$\ddot{e}(t) + K_{D}\dot{e}(t) + K_{P}e(t) = 0.$$

Characteristic equation:

$$\lambda^2 + K_D \lambda + K_P = 0$$

Stable for all positive *K*_{*D*}, *K*_{*P*} values! - no integral term

- two parameter to tune

$$\lambda_{1,2} = -\frac{1}{2}K_D \pm \frac{1}{2}\sqrt{K_D^2 - 4K_P}$$
$$e(t) = C_1 e^{\lambda_1(t)} + C_2 e^{\lambda_2(t)}$$

Closed-loop CTC (alternative method)



Closed-loop CTC (alternative method)

Closed-loop CTC (alternative method, stability)



Stable for all positive Λ values!

Nominal trajectory generation (if there is no pre-defined one)

• Simple logical condition for the reference signal:

$$r_0(t) = \begin{cases} 0.15 \text{ if } t < 1.3\\ 0.0 \text{ otherwise,} \end{cases}$$

• Reference "pre-filtering" to get smooth nominal tarjectory:

$$\xrightarrow{r_0(t)} T_1 \xrightarrow{r_1(t)} T_1 \xrightarrow{r_2(t)} T_1 \xrightarrow{r_3(t)} T_1 \xrightarrow{r_4(t)} T_1$$

• State-space representation of the trajectory generation process:

$$\begin{bmatrix} \dot{r}_1(t) \\ \dot{r}_2(t) \\ \dot{r}_3(t) \\ \dot{r}_4(t) \end{bmatrix} = \begin{bmatrix} -\frac{1}{\tau} & 0 & 0 & 0 \\ \frac{1}{\tau} & -\frac{1}{\tau} & 0 & 0 \\ 0 & \frac{1}{\tau} & -\frac{1}{\tau} & 4 \\ 0 & 0 & \frac{1}{\tau} & -\frac{1}{\tau} \end{bmatrix} \begin{bmatrix} r_1(t) \\ r_2(t) \\ r_3(t) \\ r_4(t) \end{bmatrix} + \begin{bmatrix} \frac{1}{\tau} \\ 0 \\ 0 \\ 0 \end{bmatrix} r_0(t).$$

• Smooth velocity and accelaration curves:



System identification

• "mbK" model:

$$m_{red}\ddot{q}(t) + b_{red}\dot{q}(t) = Ku(t)$$
$$\begin{bmatrix} \ddot{q}(t) \\ \dot{q}(t) \end{bmatrix} = \begin{bmatrix} \frac{b_{red}}{m_{red}} & 0 \\ 1 & 0 \end{bmatrix} \begin{bmatrix} \dot{q}(t) \\ q(t) \end{bmatrix} + \begin{bmatrix} \frac{K}{m_{red}} \\ 0 \end{bmatrix} u(t)$$

• "RLBK" model:

 $L\dot{I}(t) + RI(t) + K_e \dot{q}(t) = u(t)$ $m_{red}\ddot{q}(t) + b_{red}\dot{q}(t) - K_m I(t) = 0$

$$\begin{bmatrix} \dot{I}(t) \\ \ddot{q}(t) \\ \dot{q}(t) \end{bmatrix} = \begin{bmatrix} -\frac{R}{L} & -\frac{K_e}{L} & 0 \\ \frac{K_m}{m_{red}} & -\frac{b_{red}}{m_{red}} & 0 \\ 0 & 1 & 0 \end{bmatrix} \begin{bmatrix} I(t) \\ \dot{q}(t) \\ q(t) \end{bmatrix} + \begin{bmatrix} \frac{1}{L} \\ 0 \\ 0 \end{bmatrix} u(t)$$



State estimator for filtering out position and velocity signal noise

$$\begin{bmatrix} \ddot{\tilde{q}}(t) \\ \dot{\tilde{q}}(t) \end{bmatrix} = \begin{bmatrix} \frac{b_{red}}{m_{red}} & 0 \\ 1 & 0 \end{bmatrix} \begin{bmatrix} \ddot{\tilde{q}}(t) \\ \tilde{\tilde{q}}(t) \end{bmatrix} + \begin{bmatrix} \frac{K}{m_{red}} \\ 0 \end{bmatrix} u(t) + \begin{bmatrix} L_1 \\ L_2 \end{bmatrix} (q - \tilde{q})$$

$$\cdot \text{ ,,mbK" model:}$$

$$u^{Des} = \frac{1}{K} (m_{red} \ddot{q}^{Des}(t) + b_{red} \ddot{\tilde{q}}(t))$$

Full realized CTC scheme



Trajectory tracking for $\Lambda = 1...15$





Integral error, $\Lambda = 7...15$



Trajectory tracking for $\Lambda = 1..15$



Best trajectory tracking results $\Lambda = 12..13$



Possible explanations for oscillations at high Λ values:



Demonstration at different events:

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Kutatók éjszakája



OE Állásbörze



Thank you for your attention!